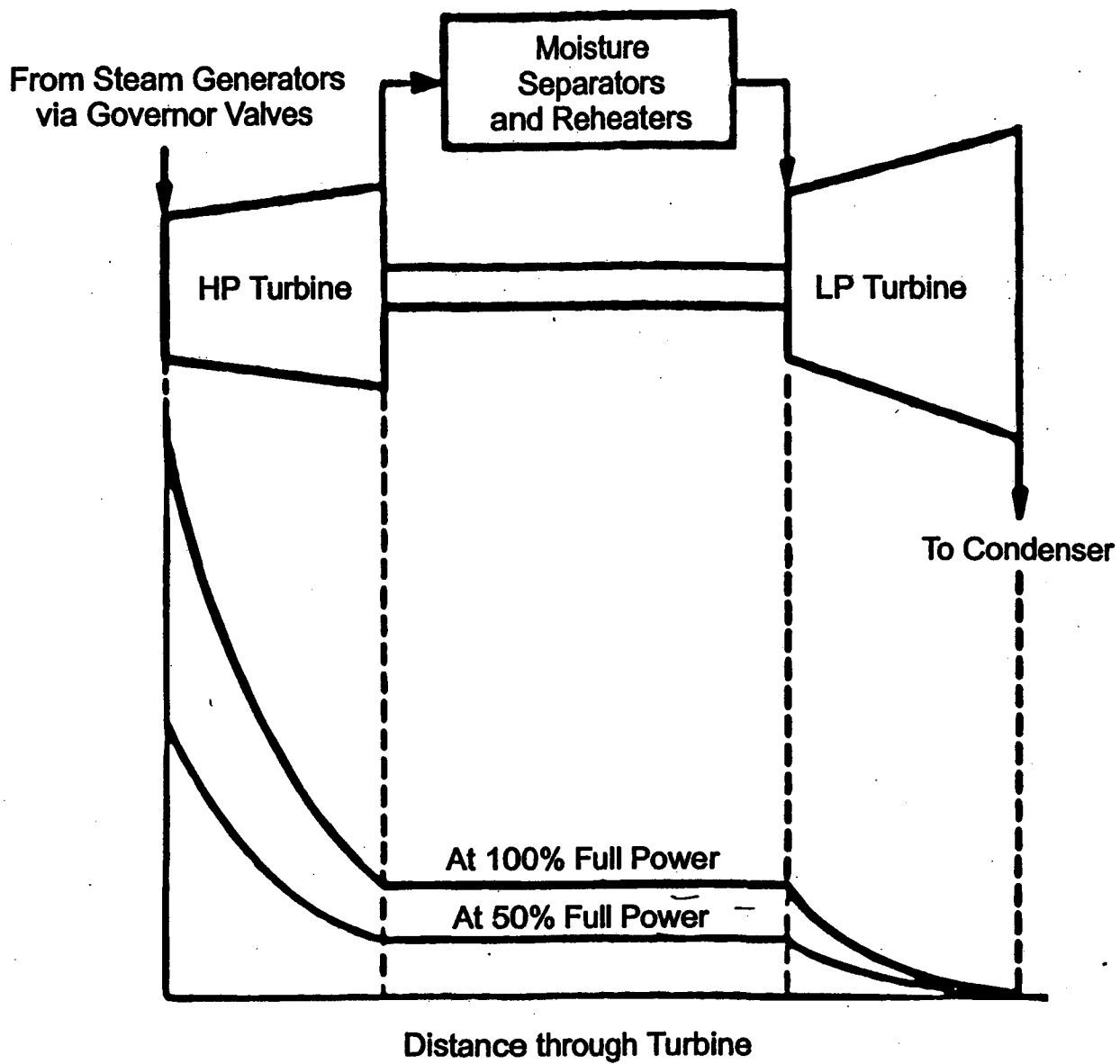


## **LECTURE 8**

# **TURBINE OPERATION PART LOAD OPERATION**



**Figure 14 Steam pressure variation through turbine**

PRESSURE GRADIENT THROUGH A TURBINE UNIT

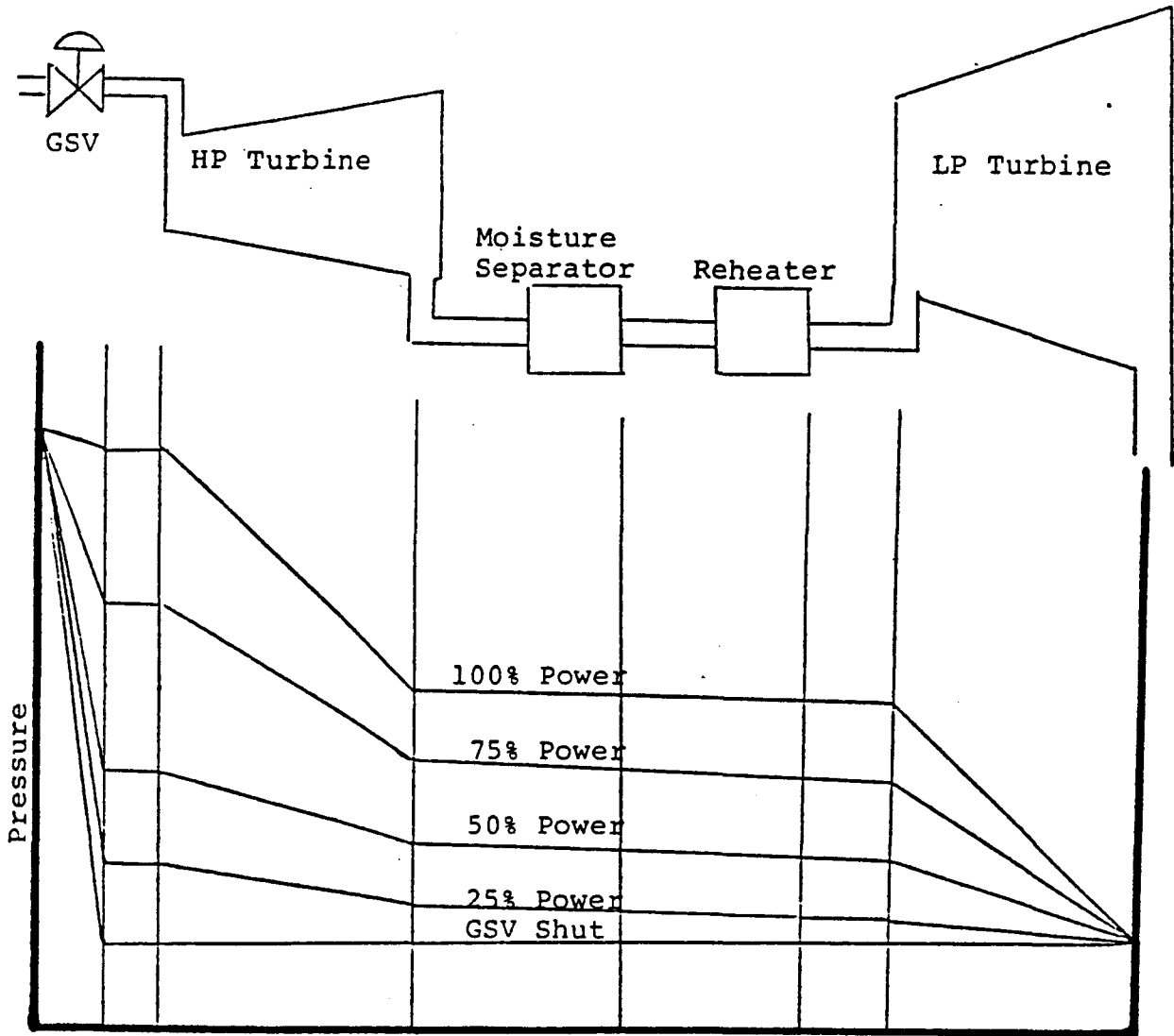
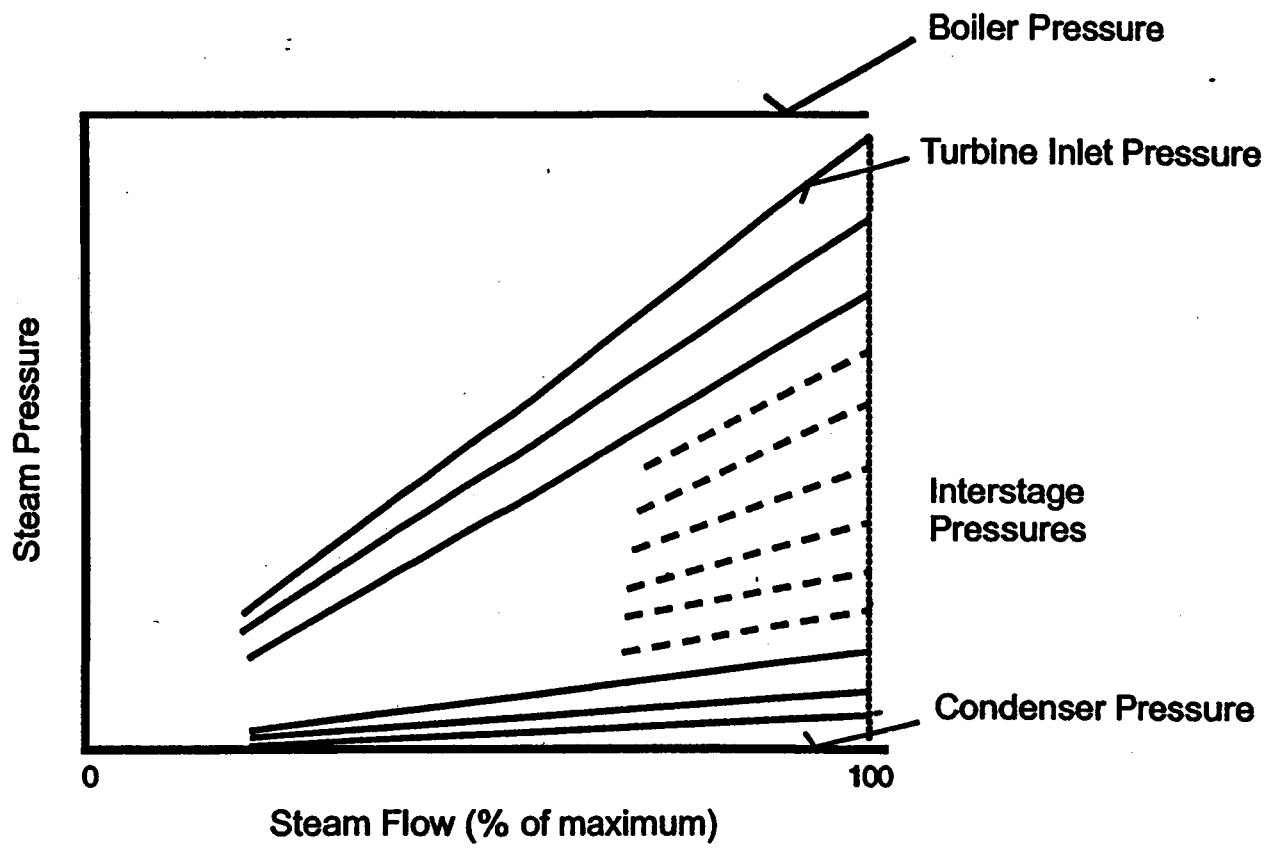


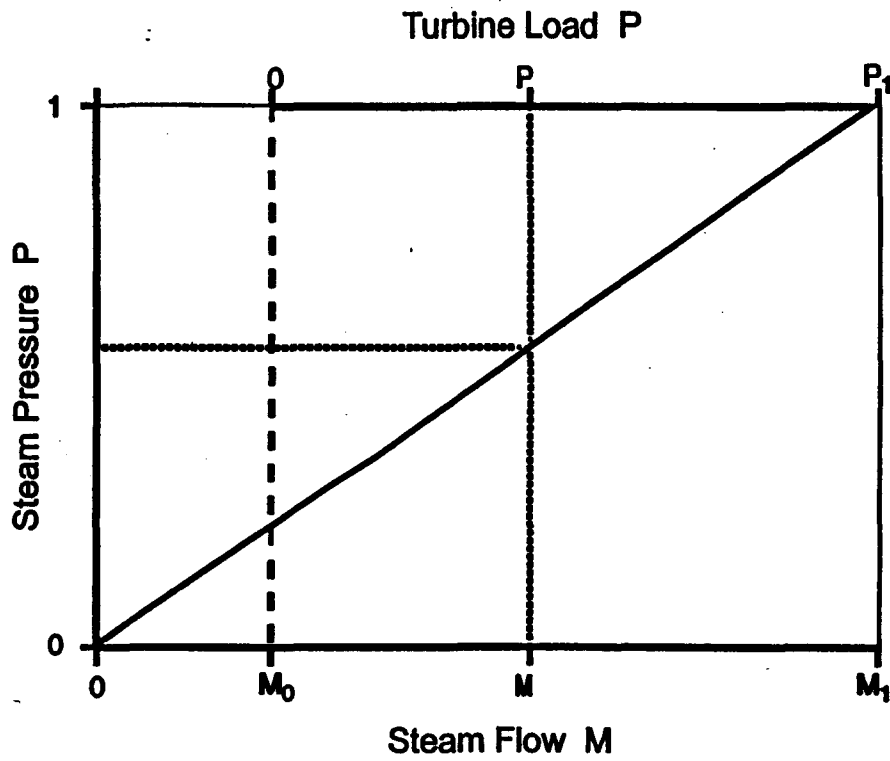
Figure 1.16

**FIGURE 6.37**

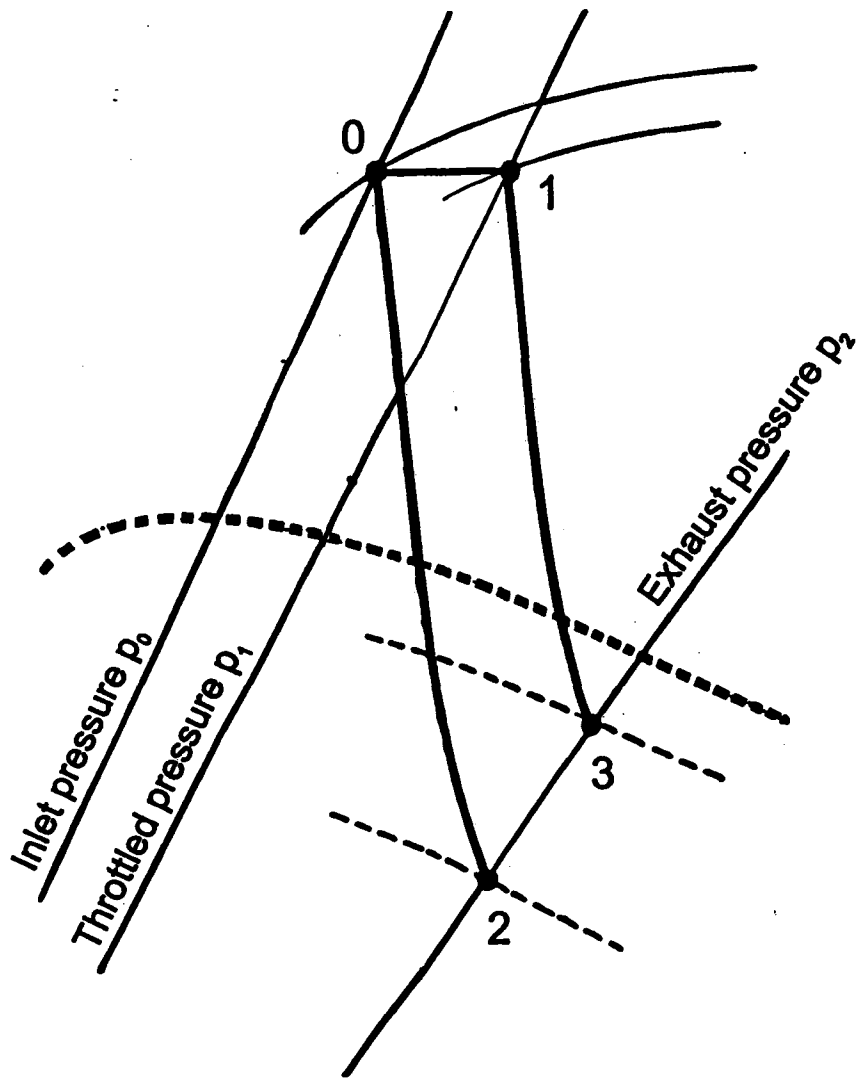
**FIGURE 7.19**



**Figure 15 Steam pressure variation with load change**



**Figure 16 Steam pressure - steam flow - turbine load relationship**



**Figure 17 Effect of throttling on turbine expansion line**

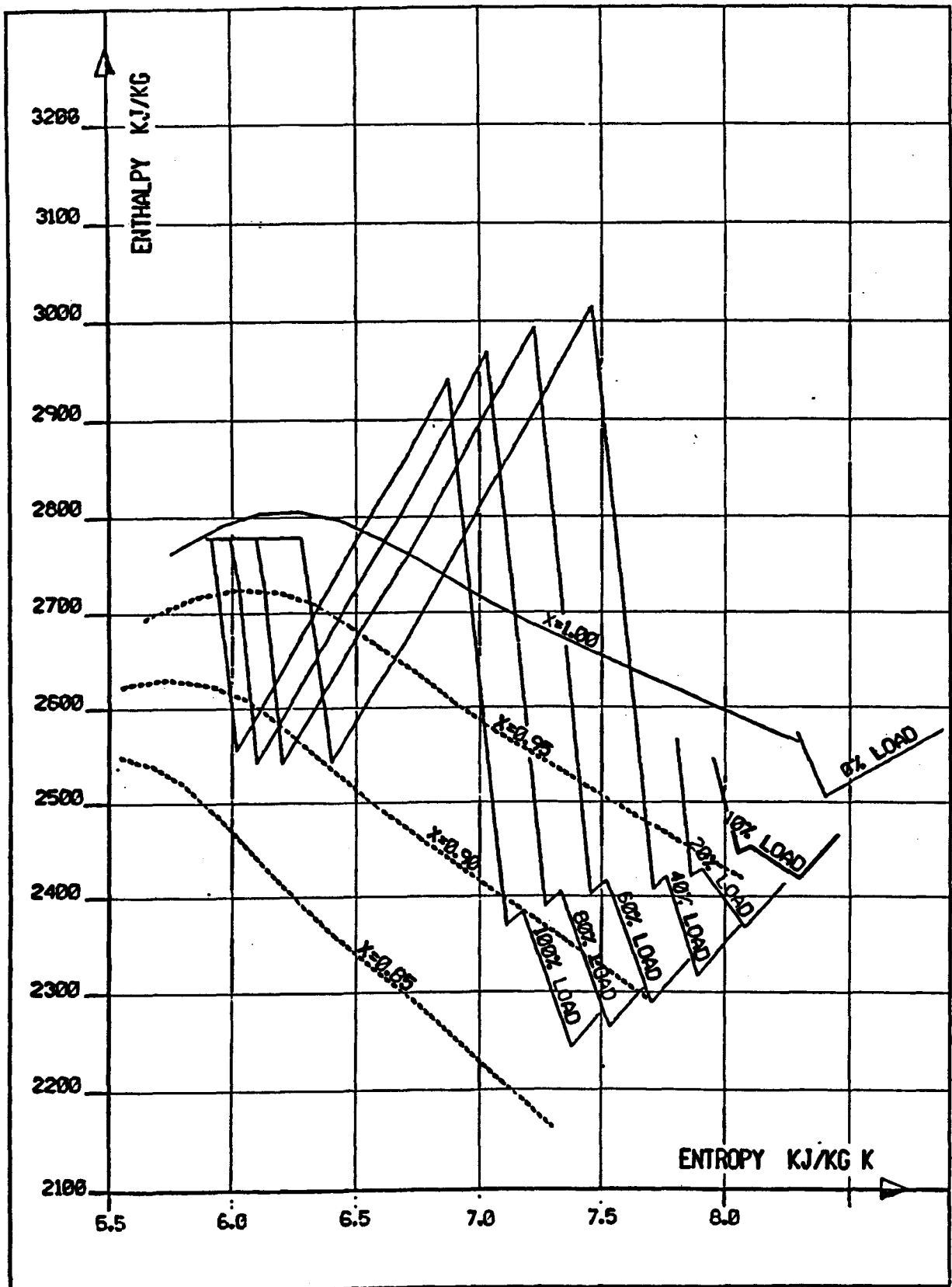
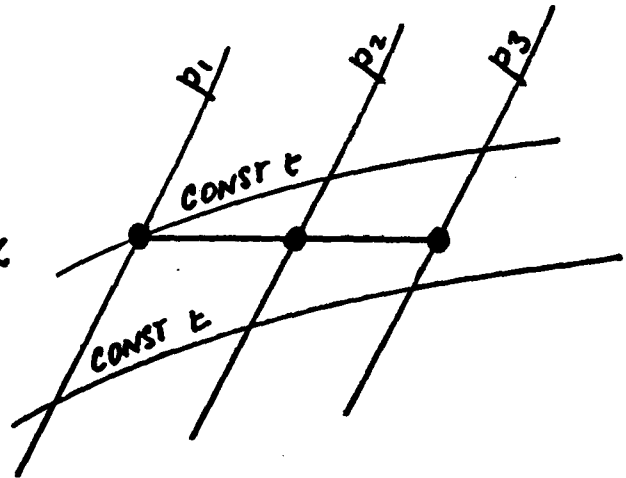


Figure 18 Part load turbine expansion lines (adapted courtesy of Eskom)

# STEAM FLOW IN BLADING

## MOLLIER CHART

THROTTLING AT GOVERNOR  
VALVE IS AT NEARLY  
CONSTANT TEMPERATURE



$$\therefore p_1 v_1 \approx p_2 v_2 \quad (n \approx 1)$$

## TURBINE FLOW (REDUCED BY THROTTLING)

$$\text{IF } p_2 = \frac{1}{2} p_1$$

$$v_2 = 2 v_1$$

$$M_2 = \frac{1}{2} M_1 \quad \text{SINCE MASS FLOW } M \propto p$$

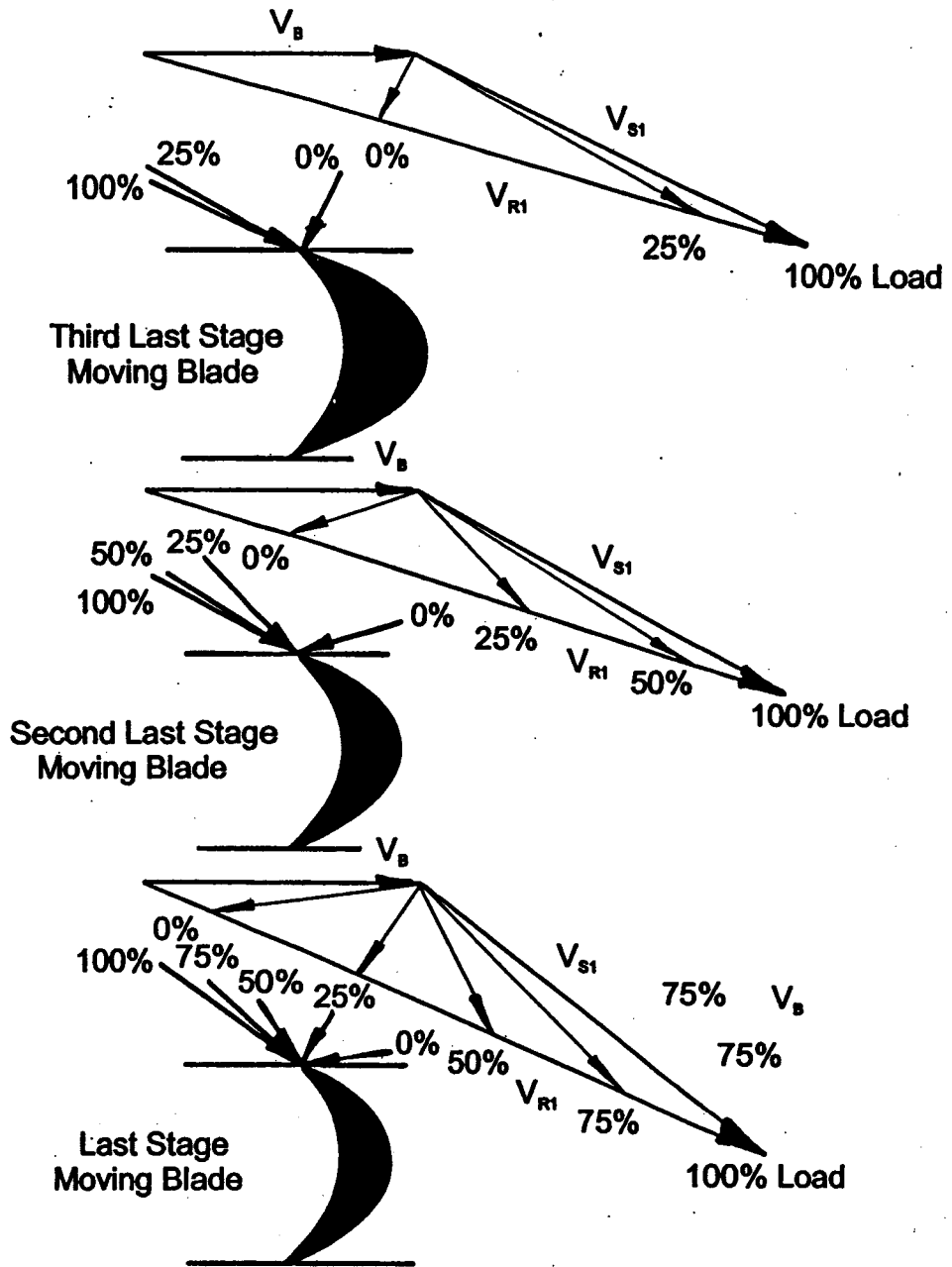
$$p_2 v_2 A_2 = \frac{1}{2} p_1 v_1 A_1 \quad (\text{SEPARATE CONDITIONS})$$

$$v_1 v_2 = \frac{1}{2} v_2 v_1 \quad \text{SINCE AREA IS THE SAME}$$

$$v_1 v_2 = \frac{1}{2} (2 v_1) v_1$$

$$v_2 = v_1$$

VELOCITY OF STEAM IS UNCHANGED



**Figure 19 Variation in velocity diagrams at various loads (adapted from Kearton)**

# Velocity Diagrams at Various Loads

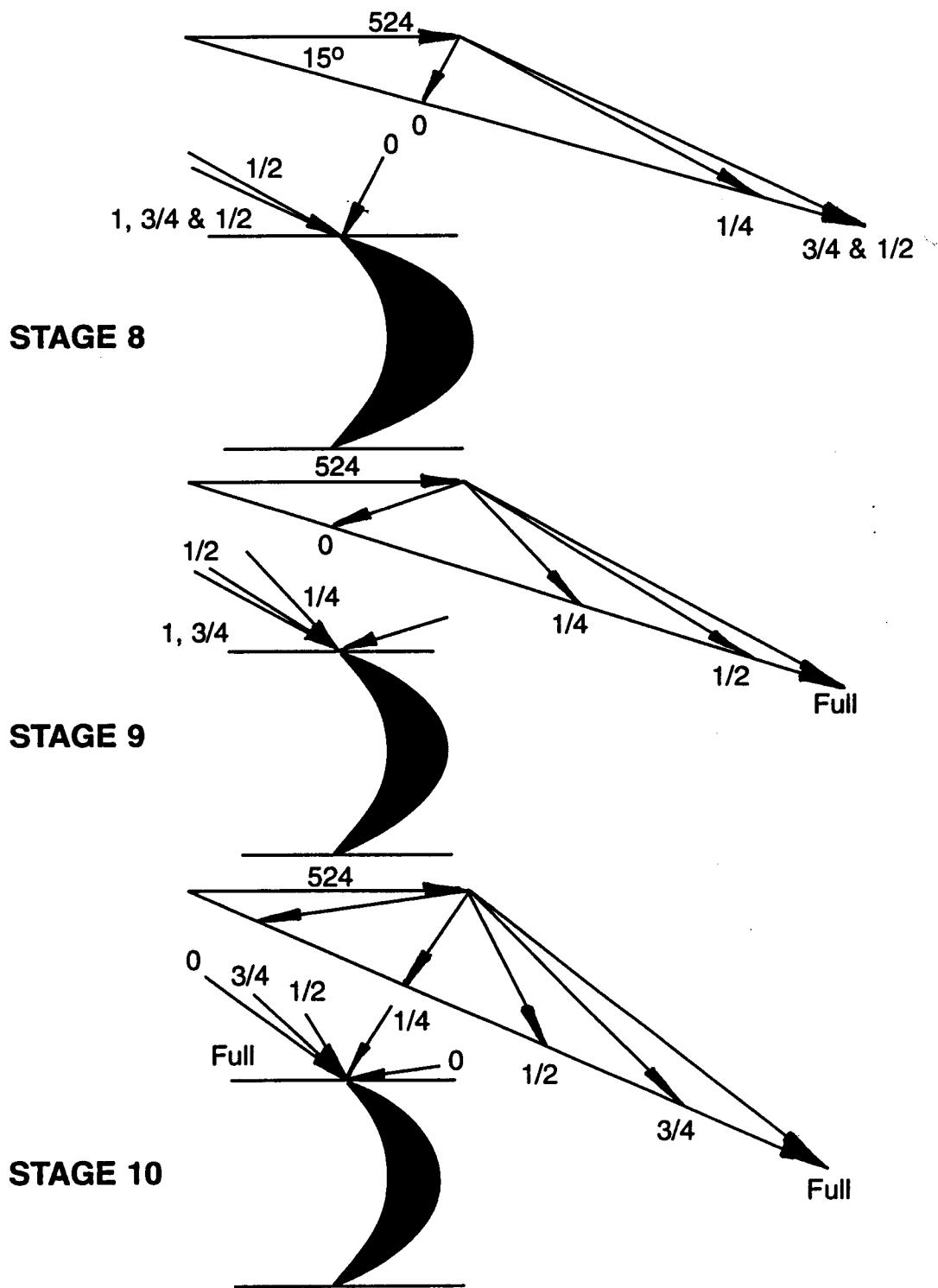
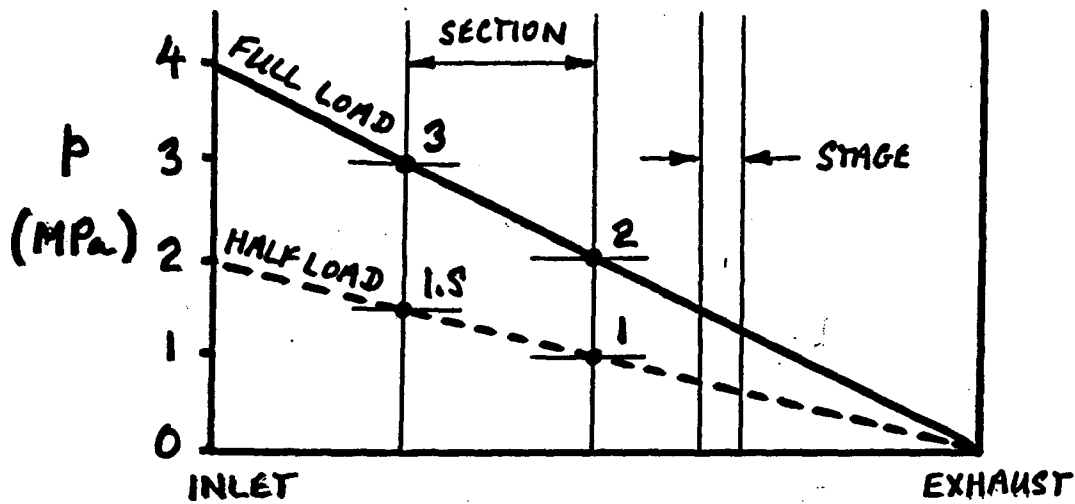


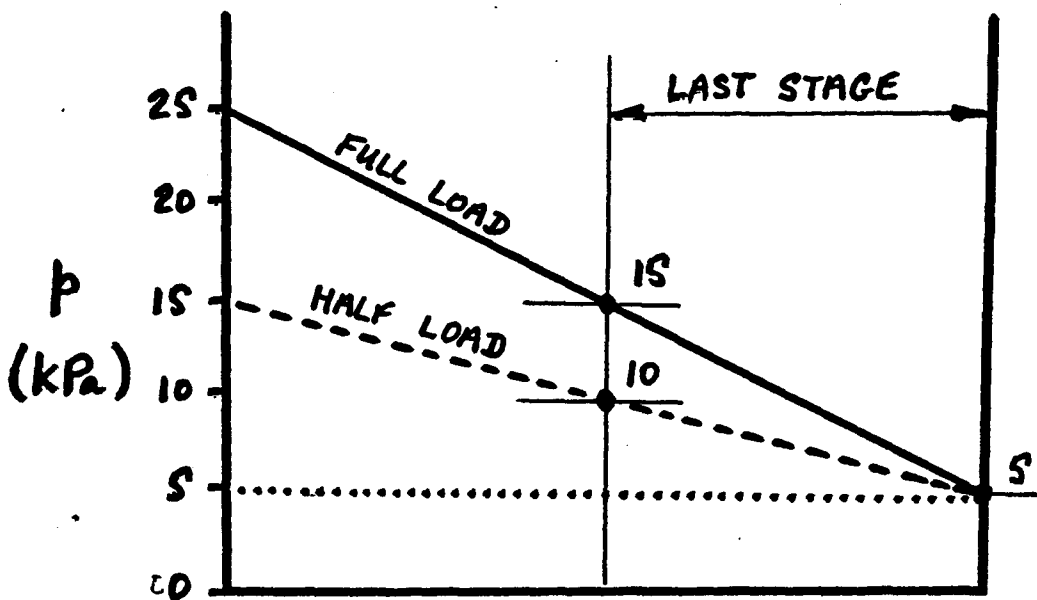
Figure 1.29

# LAST STAGE CONDITIONS



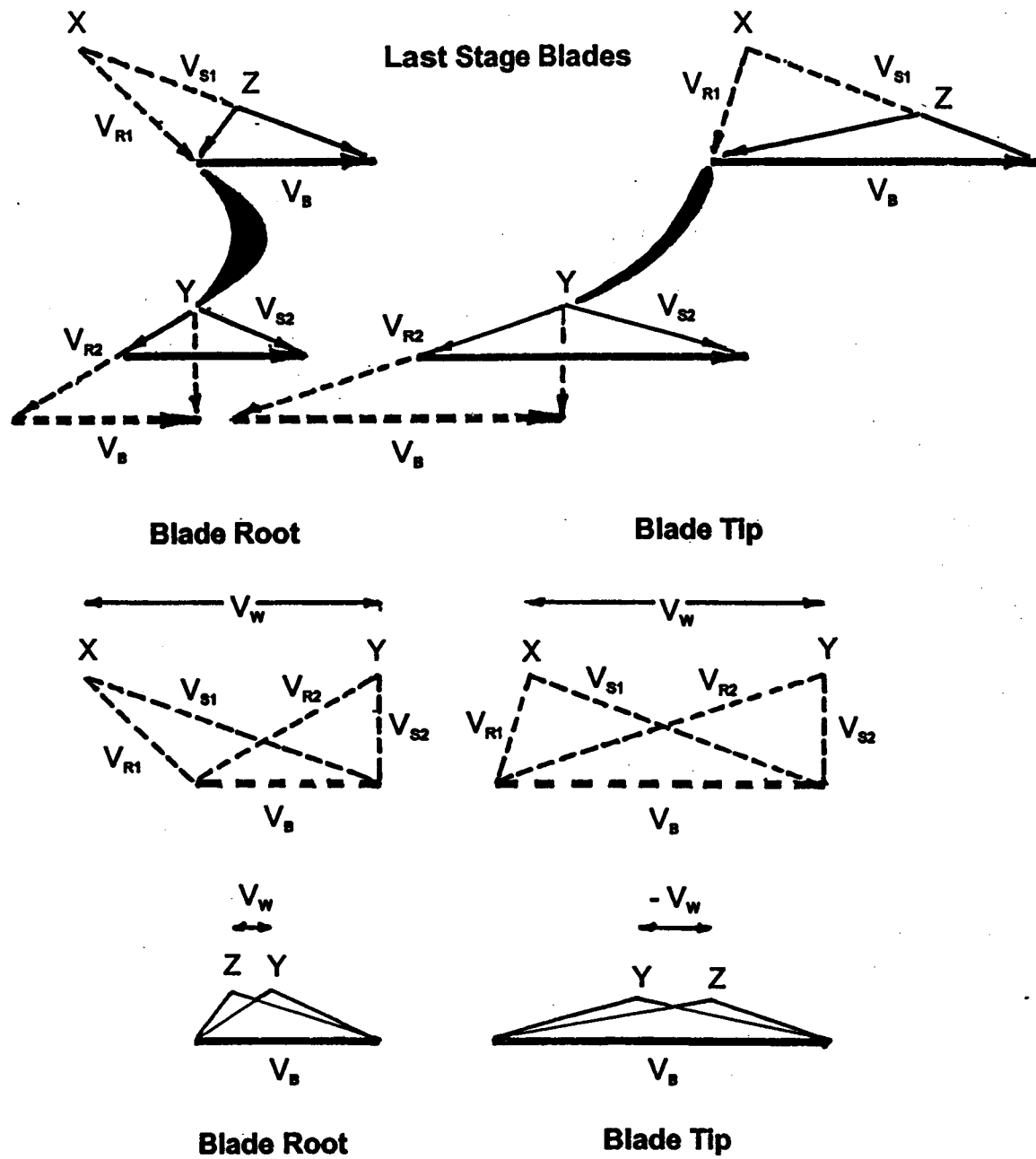
PRESSURE RATIO AT FULL LOAD =  $\frac{3}{2} = 1.5$

PRESSURE RATIO AT HALF LOAD =  $\frac{1.5}{1} = 1.5$

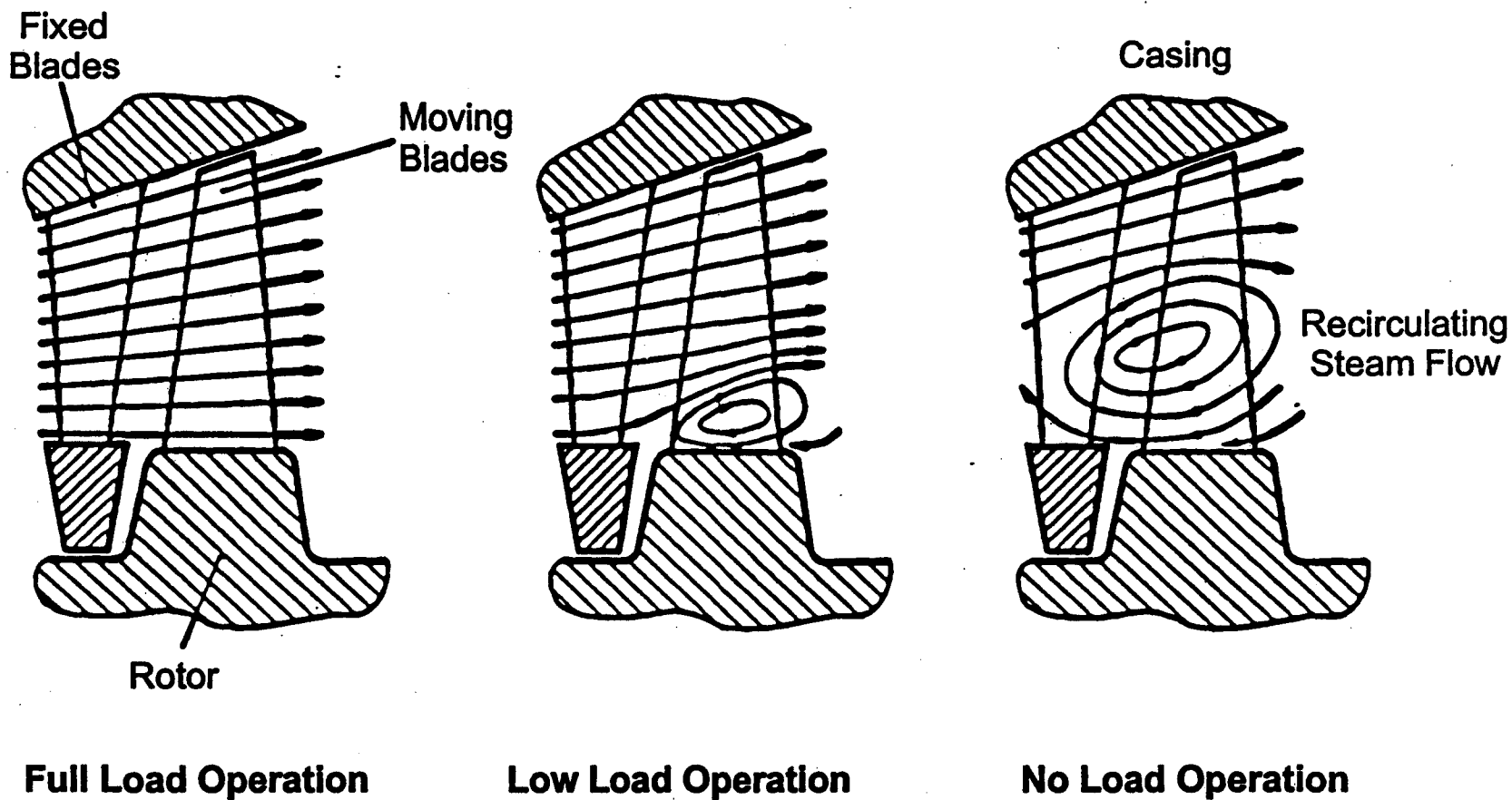


PRESSURE RATIO AT FULL LOAD =  $\frac{15}{5} = 3$

PRESSURE RATIO AT HALF LOAD =  $\frac{10}{5} = 2$



**Figure 20 Change in whirl velocity at low loads**



**Figure 21 Recirculation of steam at low loads (courtesy of NB Power)**

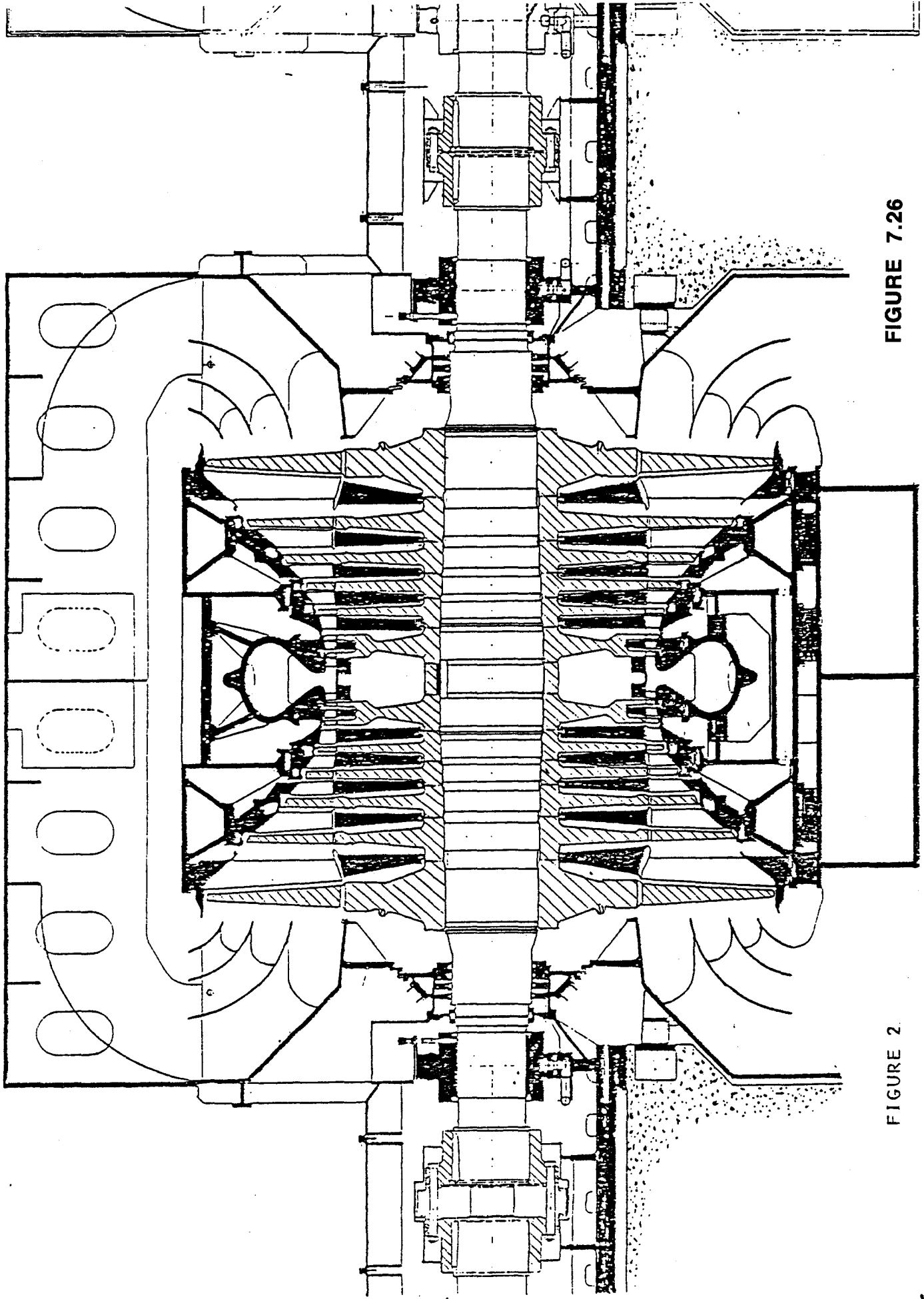


FIGURE 7.26

FIGURE 2